

Control Laboratory Projects

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State-Space Estimator-Controller Design

State-Space Controller Design

$$\begin{cases} \dot{\mathbf{x}} = \mathbf{A} \cdot \mathbf{x} + \mathbf{B} \cdot u, & u = -\mathbf{K} \cdot \mathbf{x} \\ \mathbf{K} = \text{place}(\mathbf{A}, \mathbf{B}, \mathbf{P}_c), & \mathbf{P}_c: \text{controller - eigenvalues} \end{cases}$$

State-Space Estimator Design

$$\mathbf{L} = \text{place}(\mathbf{A}^T, \mathbf{C}^T, \mathbf{P}_e)^T, \quad \mathbf{P}_e = (3 \sim 10)\mathbf{P}_c: \text{estimator - eigenvalues}, \quad \mathbf{C}: \text{available outputs}$$

$$\begin{cases} \dot{\hat{\mathbf{x}}} = (\mathbf{A} - \mathbf{L}\mathbf{C})\hat{\mathbf{x}} + \mathbf{B} \cdot u + \mathbf{L} \cdot \mathbf{y} \\ \mathbf{u} = -\mathbf{K} \cdot \hat{\mathbf{x}}; \end{cases}$$

Derivation of Time-Constant

If a velocity response is available then the time-constant is determined at 63% of the steady-state value. This section is applicable if such response is not available, a tacho-generator is out of order or does not exist. For a step input of a -amplitude

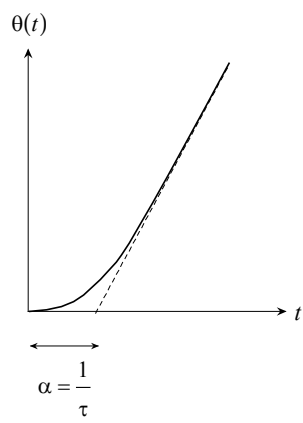
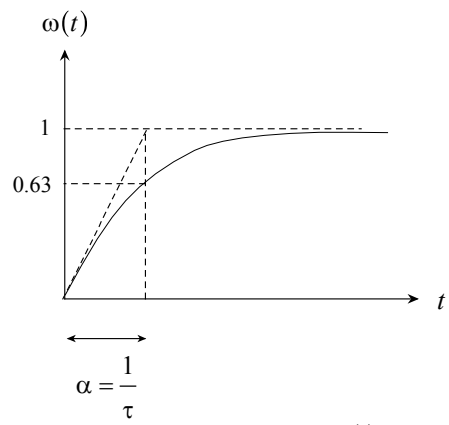
$$\hat{\theta}(s) = \frac{aK_m}{s^2(s + \alpha)}, \quad \alpha = \frac{1}{\tau}$$

or

$$\theta(t) = aK_m \int_0^t \frac{1}{\alpha} (1 - e^{-\alpha u}) du = \frac{aK_m}{\alpha^2} (\alpha t - 1 + e^{-\alpha t})$$

so its asymptote is

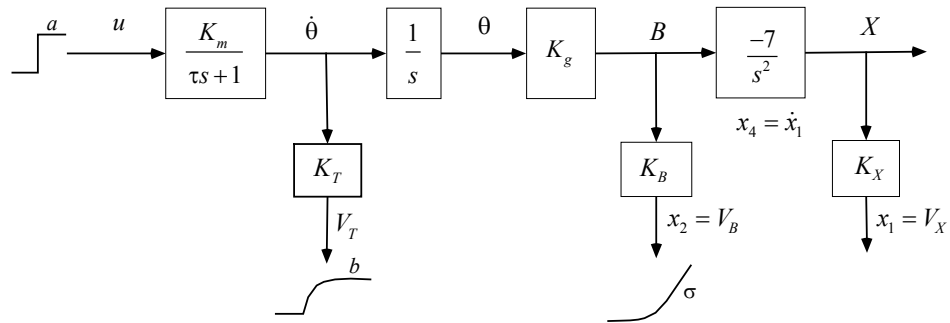
$$\varepsilon(t) = \frac{aK_m}{\alpha^2} (\alpha t - 1)$$



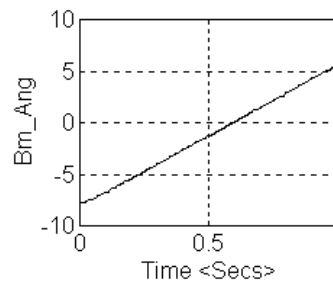
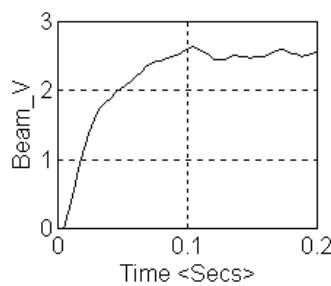
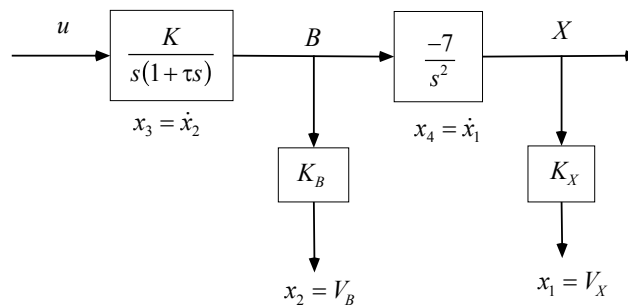
BALL-BEAM SERVO-MOTOR SYSTEM

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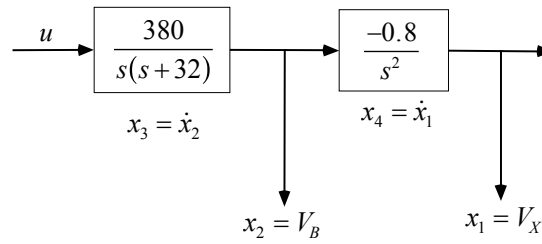
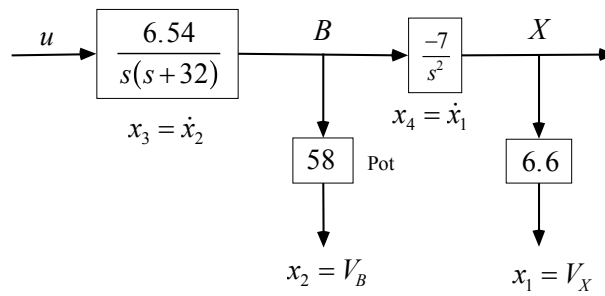


$$\frac{b}{a} = K_T K_m, \quad \frac{\sigma}{a} = K_B K_g K_m, \quad \frac{V_B}{\theta} = K_B K_g$$



Experimental data give

$$\begin{cases} \tau = 0.032 \text{ sec} \\ K_B = 57.95 \text{ V/rad (Pot)}, \quad K_X = 6.6 \text{ V/m} \\ \begin{cases} KK_B = 10.444 \sim 13.833 \\ K_T = \frac{B_{dot}}{B_{tacho}} = -2.44 \sim -2.49 \end{cases} \end{cases}$$



$$x_3 = \dot{x}_2 = K_T B_{Tacho}, \quad K_T = \frac{B_{dot}}{B_{tacho}} = -2.44 \sim -2.49$$

$$\begin{cases} \dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}u \\ \mathbf{y} = \mathbf{C}\mathbf{x} + \mathbf{D}u \end{cases}$$

$$\mathbf{A} = \begin{bmatrix} 0 & 0 & 0 & 1 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & -32 & 0 \\ 0 & -0.8 & 0 & 0 \end{bmatrix}, \quad \mathbf{B} = \begin{bmatrix} 0 \\ 0 \\ 380 \\ 0 \end{bmatrix}, \quad \mathbf{C} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \end{bmatrix}, \quad \mathbf{D} = \begin{bmatrix} 0 & 0 \\ 0 & 0 \end{bmatrix}$$

For a reasonable control effort, choose system eigenvalues as

$$\mathbf{P} = [-5, -5, -5, -5]$$

so a control gain is

$$\mathbf{K} = [-2.056, 0.395, -0.032, -1.645]$$

and an estimator is

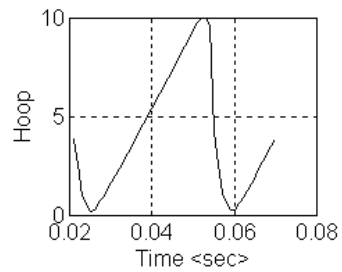
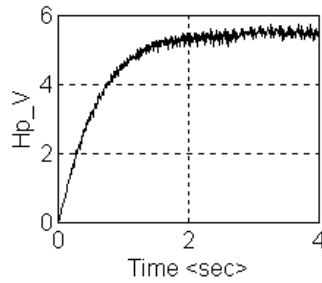
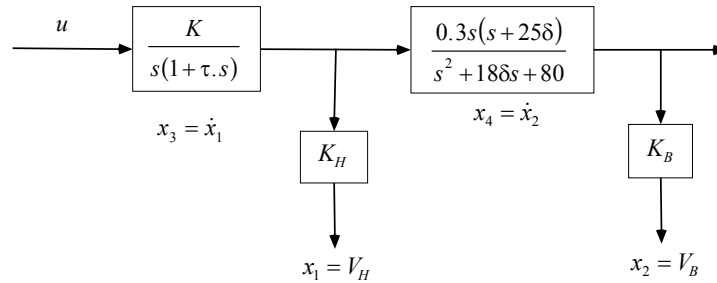
$$\mathbf{A}_e = \begin{bmatrix} -25 & 0 & 0 & 1 \\ 0 & -25 & 1 & 0 \\ 0 & 0 & -32 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}, \quad \mathbf{L} = \begin{bmatrix} 25 & 0 \\ 0 & 25 \\ 0 & 0 \\ 0 & -0.8 \end{bmatrix}$$

Remark: The allocation of 4 identical eigenvalues is similar to that of ITAE and Bessel filter, but it simplifies the controllers as above.

Ball-Hoop System

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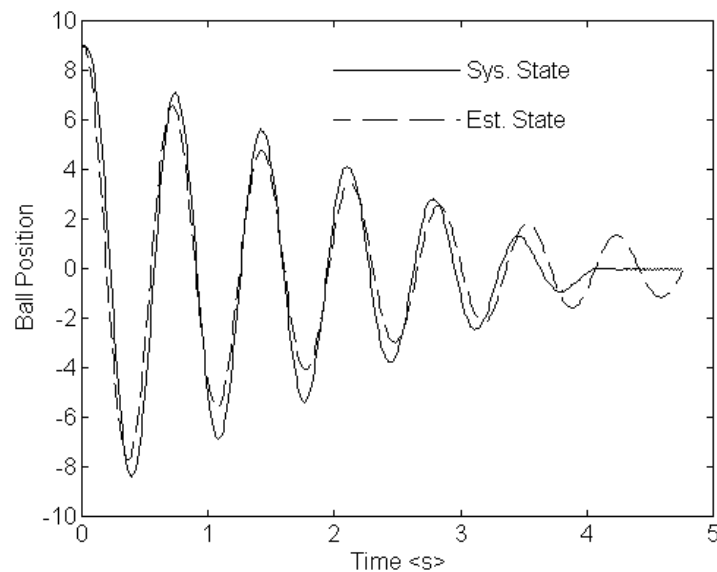
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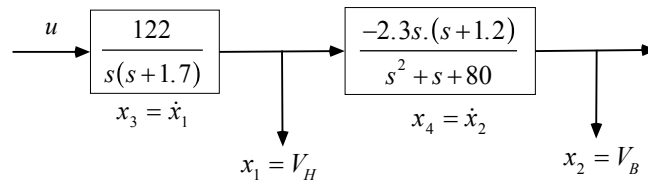
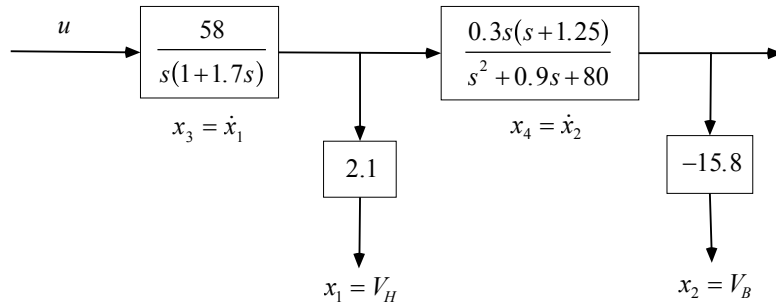


Experimental data give

$$\begin{cases} \tau = 0.568 \sim 0.6 \text{ sec} \\ K_H = 2.1 \text{ V/rad}, \quad K_B = -15.8 \text{ V/rad} \\ \begin{cases} KK_H = 65.487 \sim 77.290 \\ K_T = \frac{H_{dot}}{H_{tacho}} = 65.62 \sim 66.80 \end{cases} \end{cases}$$

With
 $\delta = 0.05$
we obtain





$$x_3 = \dot{x}_1 = K_T H_{Tacho}, \quad K_T = \frac{H_{dot}}{H_{tacho}} = 66$$

$$\begin{cases} \dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}u \\ \mathbf{y} = \mathbf{C}\mathbf{x} + \mathbf{D}u \end{cases}$$

where

$$\mathbf{A} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & -1.7 & 0 \\ 0 & -80 & 1.2 & -1 \end{bmatrix}, \quad \mathbf{B} = \begin{bmatrix} 0 \\ 0 \\ 122 \\ -281 \end{bmatrix}, \quad \mathbf{C} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \end{bmatrix}, \quad \mathbf{D} = \begin{bmatrix} 0 & 0 \\ 0 & 0 \end{bmatrix}$$

For a reasonable control effort, choose system eigenvalues as

$$\mathbf{P} = [-12, -12, -12, -12]$$

so a control gain is

$$\mathbf{K} = [2.125, -1.720, 0.609, 0.103]$$

and an estimator is

$$\mathbf{A}_e = \begin{bmatrix} -60, & 0, & 1, & 0 \\ 0, & -60, & 0, & 1 \\ 0, & 0, & -1.7, & 0 \\ 0, & 0, & 1.2, & -1 \end{bmatrix}, \quad \mathbf{L} = \begin{bmatrix} 60, & 0 \\ 0, & 60 \\ 0, & 0 \\ 0, & -80 \end{bmatrix}$$

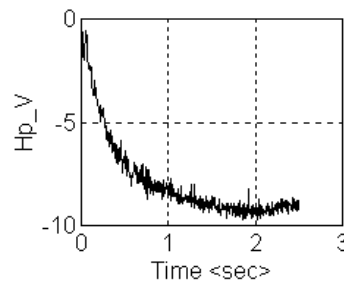
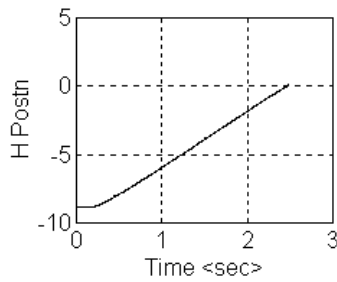
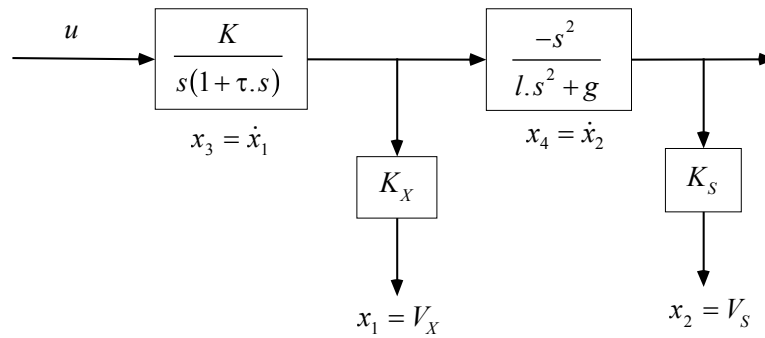
Remark: The 4 identical eigenvalues are the best, this allocation is much better than that of ITAE and Bessel filter. Particularly, with 2 identical eigenvalues, there are identical pole and zero for elimination in each controller and we have come up with the simple controllers above.

Problem: there is a common factor of s in the feedforward path, so the step response is of *zero*. It is wrong with the experimental results.

OverHead-Crane System

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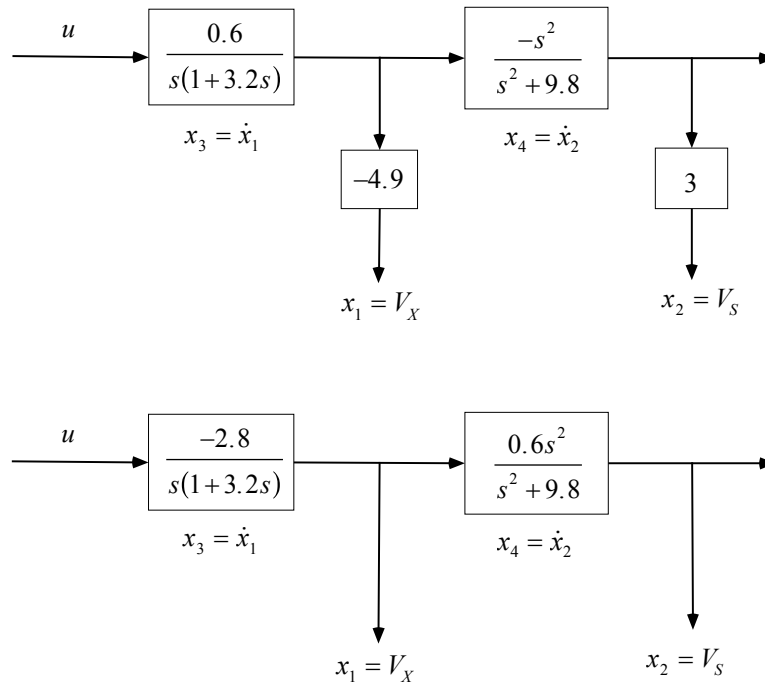
Experimental data give

$$\begin{cases} \tau = 0.305 \sim 0.32 \text{ sec} \\ K_x = -4.9 \text{ V/m}, \quad K_s = 3 \text{ V/rad} \\ \begin{cases} KK_x = -0.862 \sim -0.910 \\ K_T = \frac{H_{dot}}{H_{tacho}} = -0.438 \sim -0.46 \end{cases} \end{cases}$$

With

$$l = 1 \text{ m}, \quad g = 9.8 \text{ ms}^{-2}$$

we obtain



$$x_3 = \dot{x}_1 = K_T X_{Tacho}, \quad K_T = \frac{X_{dot}}{X_{tacho}} = -0.46$$

$$\begin{cases} \dot{\mathbf{x}} = \mathbf{Ax} + \mathbf{Bu} \\ \mathbf{y} = \mathbf{Cx} + \mathbf{Du} \end{cases}$$

where

$$\mathbf{A} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & -3.2 & 0 \\ 0 & -9.8 & 1.92 & -1 \end{bmatrix}, \quad \mathbf{B} = \begin{bmatrix} 0 \\ 0 \\ -2.8 \\ -1.68 \end{bmatrix}, \quad \mathbf{C} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \end{bmatrix}, \quad \mathbf{D} = \begin{bmatrix} 0 & 0 \\ 0 & 0 \end{bmatrix}$$

For a reasonable control effort, choose system eigenvalues as

$$\mathbf{P} = [-2.5, -2.5, -2.5, -2.5]$$

so a control gain is

$$\mathbf{K} = [-1.424, -14.115, -1.135, -2.156]$$

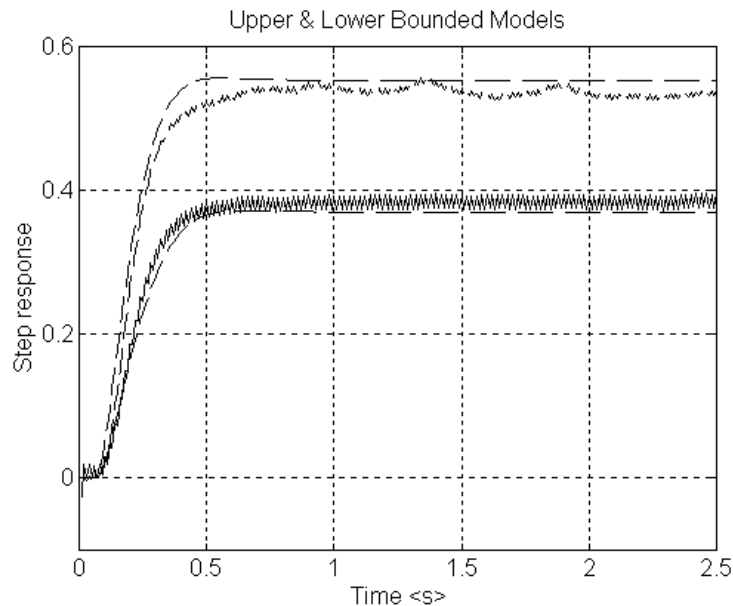
and an estimator is

$$\mathbf{A}_e = \begin{bmatrix} -12.5 & 0 & 1 & 0 \\ 0 & -12.5 & 0 & 1 \\ 0 & 0 & -3.2 & 0 \\ 0 & 0 & -1.92 & 0 \end{bmatrix}, \quad \mathbf{L} = \begin{bmatrix} 12.5 & 0 \\ 0 & 12.5 \\ 0 & 0 \\ 0 & -9.8 \end{bmatrix}$$

Static-VAR System

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From the figure above we can estimate the system model by the following upper & lower bounded models by different step responses at different torque loads

$$G_{p1}(s) = \frac{37 \cdot e^{-s \cdot T_d}}{s^2 + 17s + 100}, \quad G_{p2}(s) = \frac{83 \cdot e^{-s \cdot T_d}}{s^2 + 21s + 150}$$

and a nominal model can be chosen as

$$G_p(s) = \frac{60 \cdot e^{-s \cdot T_d}}{s^2 + 19s + 125}$$

where

$T_d = 0.05$ sec : transportation lag

If the sampling time is taken into account, a digital-equivalent model can be

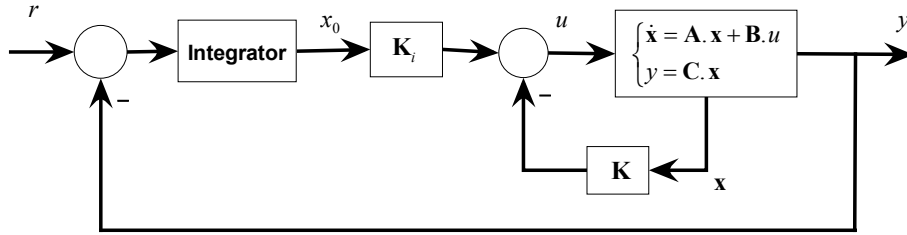
$$G_p(s) \approx \frac{60 \cdot \left(1 - T_d \cdot s + \frac{T_d^2}{2} s^2\right)}{(s^2 + 19s + 125) \left(\frac{T_s}{2} s + 1\right)}$$

if the sampling-time is ignored

$$G_p(s) \approx \frac{60 \cdot (1 - T_d \cdot s)}{s^2 + 19s + 125}$$

this model will be used in designing a controller.

Remark. the reference will be the step of 5 since the voltage 5 volts is equivalent to the unity power factor.



Let

$$x_0(t) = \int_{t_0}^t e(\tau) \cdot d\tau, \quad e(t) = y(t) - r(t)$$

be an integral of the error between the output and the reference input for the desired output, so

$$\dot{x}_0 = e = y - r = \mathbf{C}\mathbf{x} - r$$

then

$$\dot{\tilde{\mathbf{x}}} = \mathbf{A}_i \cdot \tilde{\mathbf{x}} + \mathbf{B}_i \cdot u - \mathbf{r}$$

where

$$\tilde{\mathbf{x}} = \begin{bmatrix} x_0 \\ \mathbf{x} \end{bmatrix}, \quad \mathbf{A}_i = \begin{bmatrix} 0 & \mathbf{C} \\ 0 & \mathbf{A} \end{bmatrix}, \quad \mathbf{B}_i = \begin{bmatrix} 0 \\ \mathbf{B} \end{bmatrix}, \quad \mathbf{r} = \begin{bmatrix} r \\ \mathbf{0} \end{bmatrix}$$

$$\mathbf{A}_i = \begin{bmatrix} 0 & 1 & -0.05 \\ 0 & 0 & 1 \\ 0 & -125 & -19 \end{bmatrix}, \quad \mathbf{B}_i = \begin{bmatrix} 0 \\ 0 \\ 60 \end{bmatrix}, \quad \mathbf{C} = [0 \ 1 \ -0.05], \quad \mathbf{r} = \begin{bmatrix} 5 \\ 0 \\ 0 \end{bmatrix}$$

choose the poles in s-domain at -8, we obtain the following gain matrix

$$\mathbf{K} = [8.5333 \ 1.5433 \ 0.0833]$$

For an integral state-space control, we have used

$$u = -\mathbf{K} \cdot \begin{bmatrix} \int (y-r) \cdot dt \\ y \\ \hat{x}_2 \end{bmatrix}, \quad \mathbf{K} = [8.533, \ 1.117 \ 0.083]$$

where the estimate state \hat{x}_2 is determined by the following state-space observer

$$\dot{\hat{\mathbf{x}}} = \mathbf{A}_e \cdot \hat{\mathbf{x}} + \mathbf{B} \cdot u + \mathbf{L} \cdot y, \quad \hat{\mathbf{x}} = [\hat{x}_1 \ \hat{x}_2]^T$$

with

$$\mathbf{B} = \begin{bmatrix} 0 \\ 60 \end{bmatrix}, \quad \mathbf{L} = \begin{bmatrix} 61 \\ 316 \end{bmatrix} \Rightarrow \mathbf{A}_e = \mathbf{A} - \mathbf{L}\mathbf{C} = \begin{bmatrix} -61, & 1 \\ -441, & -19 \end{bmatrix}$$